

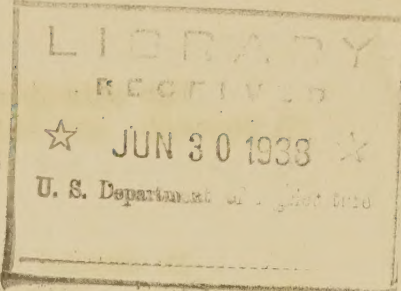
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UNITED STATES DEPARTMENT OF AGRICULTURE
Bureau of Agricultural Engineering



SIMPLIFIED PITOT TUBE CALCULATIONS OF
AIR FLOW IN DUCTS AND PIPES

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Many engineers have difficulty in making air-volume calculations in the ducts or piping systems of fans and blowers. With ordinary care, using a calibrated pitot tube and sensitive gages, it is not difficult to secure accurate readings in ducts and air pipes. If the engineer is not familiar with methods of taking such readings, he may refer to any college text book on ventilation, to the "Standard Test Code" promulgated by the American Society of Heating and Ventilating Engineers, or to certain pamphlets distributed by manufacturers of fans.

Flow of air in cubic feet per minute is calculated from the velocity-pressure or velocity-head readings of the pitot tube. The basic formula for the flow of a fluid of uniform density is

in which

$$V = \sqrt{2gh} \dots \dots \dots (1)$$

V = velocity in feet per second,
g = acceleration due to gravity (32.16 ft. per second per second), and
h = head in feet of the fluid for which V is being computed.

For a gas (air is a gas) h in formula 1 may be considered the same as the height of a column of fluid of the same density as the gas being measured that will exert a pressure equal to the pressure causing the flow. By the introduction of a proper coefficient, V and h can be expressed in any desired units.

To adapt this formula to the measurement of air flow, let

V' = velocity of air in feet per minute;
h' = head in inches of water;
W = weight per cubic foot of atmosphere at the test conditions of temperature, humidity, and barometric pressure;
 $k = \sqrt{1/W} = \sqrt{\text{cubic feet per pound of atmosphere.}}$

Then formula 1 may be written

$$V' = 1096.2 \ k \sqrt{h'} \dots \dots \dots (2)$$

For atmospheric temperature of 70 degrees F., relative humidity of 70 percent, and barometric pressure of 30 inches of mercury, this becomes

$$V' = 4012\sqrt{h'} \dots \dots \dots (3)$$

It should be kept in mind that W is the weight of one cubic foot of atmosphere. Atmosphere is composed of dry air and water vapor. The higher the relative humidity, the greater will be the velocity under conditions of constant velocity head. Values for W may be obtained from the tables of the United States Weather Bureau, or may be calculated. Figure 1 gives the cubic feet of atmosphere per pound, the reciprocal of W, under many conditions of temperature, at 30 inches barometric pressure.

For any atmospheric condition, k may be calculated as the square root of the volume in cubic feet determined from figure 1, for use in these formulas.

From many tests, the United States Navy has found that the mean velocity for an entire duct (round or its equivalent) is 91 percent of the velocity at the center of the pipe; hence

$$V_a = 997.5 k \sqrt{h_v} \dots \dots \dots (4)$$

where V_a = mean velocity of air in feet per minute

h_v = velocity head at center of pipe.

In cotton-gin work most of the readings will be taken with a pitot tube at the center of the pipe. Table 1 gives values for 997.5 k.

Table 1.- Values of 997.5 k in formula $V_a = 997.5 k \sqrt{h_v}$
(Barometric pressure, 30.0 inches)

Atmosphere conditions			:	997.5 k
Temperature (F.)		Relative humidity	:	
Degrees	:	Percent	:	
	:	(0	:	3609
60	:	(50	:	3615
	:	(100	:	3621
	:	(0	:	3645
70	:	(50	:	3653
	:	(100	:	3661
	:	(0	:	3679
80	:	(50	:	3691
	:	(100	:	3703

Graphs from which can be read the mean velocity in feet per minute of the atmosphere within the pipe will enable an engineer to make the entire calculation on the spot, concurrent with the pitot tube readings. Figure 2 was prepared for cotton ginning conditions in the Mississippi Delta region, and is based upon 70 degrees (F.) temperature, 70 percent

relative humidity, and 30 inches barometric pressure. It has been found to be sufficiently accurate for the entire cotton belt, and is correct within one percent for all humidities from 0 to 100 percent at a temperature of 70 degrees.

By use of figure 2 difficulties with pitot tube readings are eliminated and the engineer can make complete findings on the test grounds. If his results show that the rates of flow per minute are not what he believes they should be, he can find this out at once and look for causes of the trouble. With such charts it is easy to instruct assistants in making accurate observations.

In making field calculations by this method, the first step is to obtain the velocity head, h_v , by use of the pitot tube. To insure accuracy, the pitot tube must be inserted in a straight section of pipe where the instrument will be as free as possible from effects of cavitation and eddies in the air flow. Standardized fan tests require that there must be $7\frac{1}{2}$ diameters of pipe as a straight length to the point of insertion of the pitot tube, but, in cotton ginning installations, it is frequently not possible to obtain more than 4 diameters in a straight section at points where instruments may be set up. The pitot tube may be placed on either the suction or discharge side of the fan, but the discharge side is to be preferred where convenient.

The second step is that of making the necessary interpolation from figure 2 to determine the mean velocity for the given velocity head, h_v .

The third step is computation of the cubic feet per minute flowing through the pipe. This may be obtained by direct calculation, in which the mean velocity V_a is multiplied by the area of the pipe in square feet, which may be obtained from table 2; or the rate of flow may be approximated by using the chart in figure 3.

Table 2.- Cross-sectional area of cotton-gin piping

Diameter of pipe		Cross-sectional area of pipe
Inches		Square feet
8		0.3491
9		0.4418
10		0.5454
11		0.6600
$11\frac{1}{2}$		0.7215
12		0.7854
$12\frac{1}{2}$		0.8522
13		0.9218
$13\frac{1}{2}$		0.9940
14		1.069
$14\frac{1}{2}$		1.147
15		1.227
16		1.396

Cotton ginning practices usually require mean velocities of 2,000 to 4,500 feet per minute for handling seed cotton, and from 4,000 to 6,000 for handling cottonseed. Air-blast gins usually require from 155 to 175 cubic feet of air per 10 saws, and cotton unloading fans that do not blow seed have successfully operated with flows ranging from 600 to 900 cubic feet per minute per gin stand.

At times in cotton ginning, especially in cotton drying work, it is desirable to ascertain what change occurs in relative humidity when atmosphere at one particular condition is heated from one temperature to another. This may be determined from figure 4. (Reproduced from U.S.D.A. Weather Bureau Bulletin No. 235).

Table 3 will be found useful in determining the relative humidity from the readings of wet and dry bulb thermometers. This table has been compiled from Weather Bureau Bulletin 235 and Department Bulletin 1136.

To use the table: Suppose the dry bulb reads 140° and the wet bulb 130°, a difference of 10°. Find 140 in the left-hand column; follow across to the column headed 10; the relative humidity is 75 percent.

Table 3.- Relative Humidity in percent
(Pressure = 30 inches)

Difference Between Wet and Dry Bulb Thermometers in Degrees Fahrenheit																																								
Air Temp. Dry Bulb Reading Degrees Fahrenheit	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32	33	34	35	36	37	38	39	40
50	51	52	53	54	55	56	57	58	59	60	61	62	63	64	65	66	67	68	69	70	71	72	73	74	75	76	77	78	79	80	81	82	83	84	85	86	87	88	89	90
91	92	93	94	95	96	97	98	99	100	101	102	103	104	105	106	107	108	109	110	111	112	113	114	115	116	117	118	119	120	121	122	123	124	125	126	127	128	129	130	
131	132	133	134	135	136	137	138	139	140	141	142	143	144	145	146	147	148	149	150	151	152	153	154	155	156	157	158	159	160	161	162	163	164	165	166	167	168	169	170	
171	172	173	174	175	176	177	178	179	180	181	182	183	184	185	186	187	188	189	190	191	192	193	194	195	196	197	198	199	200	201	202	203	204	205	206	207	208	209	210	
211	212	213	214	215	216	217	218	219	220	221	222	223	224	225	226	227	228	229	230	231	232	233	234	235	236	237	238	239	240	241	242	243	244	245	246	247	248	249	250	
251	252	253	254	255	256	257	258	259	260	261	262	263	264	265	266	267	268	269	270	271	272	273	274	275	276	277	278	279	280	281	282	283	284	285	286	287	288	289	290	
291	292	293	294	295	296	297	298	299	300	301	302	303	304	305	306	307	308	309	310	311	312	313	314	315	316	317	318	319	320	321	322	323	324	325	326	327	328	329	330	
331	332	333	334	335	336	337	338	339	340	341	342	343	344	345	346	347	348	349	350	351	352	353	354	355	356	357	358	359	360	361	362	363	364	365	366	367	368	369	370	
371	372	373	374	375	376	377	378	379	380	381	382	383	384	385	386	387	388	389	390	391	392	393	394	395	396	397	398	399	400	401	402	403	404	405	406	407	408	409	410	
411	412	413	414	415	416	417	418	419	420	421	422	423	424	425	426	427	428	429	430	431	432	433	434	435	436	437	438	439	440	441	442	443	444	445	446	447	448	449	450	
451	452	453	454	455	456	457	458	459	460	461	462	463	464	465	466	467	468	469	470	471	472	473	474	475	476	477	478	479	480	481	482	483	484	485	486	487	488	489	490	
491	492	493	494	495	496	497	498	499	500	501	502	503	504	505	506	507	508	509	510	511	512	513	514	515	516	517	518	519	520	521	522	523	524	525	526	527	528	529	530	
531	532	533	534	535	536	537	538	539	540	541	542	543	544	545	546	547	548	549	550	551	552	553	554	555	556	557	558	559	560	561	562	563	564	565	566	567	568	569	570	
571	572	573	574	575	576	577	578	579	580	581	582	583	584	585	586	587	588	589	590	591	592	593	594	595	596	597	598	599	600	601	602	603	604	605	606	607	608	609	610	
611	612	613	614	615	616	617	618	619	620	621	622	623	624	625	626	627	628	629	630	631	632	633	634	635	636	637	638	639	640	641	642	643	644	645	646	647	648	649	650	
651	652	653	654	655	656	657	658	659	660	661	662	663	664	665	666	667	668	669	670	671	672	673	674	675	676	677	678	679	680	681	682	683	684	685	686	687	688	689	690	
691	692	693	694	695	696	697	698	699	700	701	702	703	704	705	706	707	708	709	710	711	712	713	714	715	716	717	718	719	720	721	722	723	724	725	726	727	728	729	730	
731	732	733	734	735	736	737	738	739	740	741	742	743	744	745	746	747	748	749	750	751	752	753	754	755	756	757	758	759	760	761	762	763	764	765	766	767	768	769	770	
771	772	773	774	775	776	777	778	779	780	781	782	783	784	785	786	787	788	789	790	791	792	793	794	795	796	797	798	799	800	801	802	803	804	805	806	807	808	809	810	
811	812	813	814	815	816	817	818	819	820	821	822	823	824	825	826	827	828	829	830	831	832	833	834	835	836	837	838	839	840	841	842	843	844	845	846	847	848	849	850	
851	852	853	854	855	856	857	858	859	860	861	862	863	864	865	866	867	868	869	870	871	872	873	874	875	876	877	878	879	880	881	882	883	884	885	886	887	888	889	890	
891	892	893	894	895	896	897	898	899	900	901	902	903	904	905	906	907	908	909	910	911	912	913	914	915	916	917	918	919	920	921	922	923	924	925	926	927	928	929	930	
931	932	933	934	935	936	937	938	939	940	941	942	943	944	945	946	947	948	949	950	951	952	953	954	955	956	957	958	959	960	961	962	963	964	965	966	967	968	969	970	
971	972	973	974	975	976	977	978	979	980	981	982	983	984	985	986	987	988	989	990	991	992	993	994	995	996	997	998	999	1000	1001	1002	1003	1004	1005	1006	1007	1008	1009	1010	

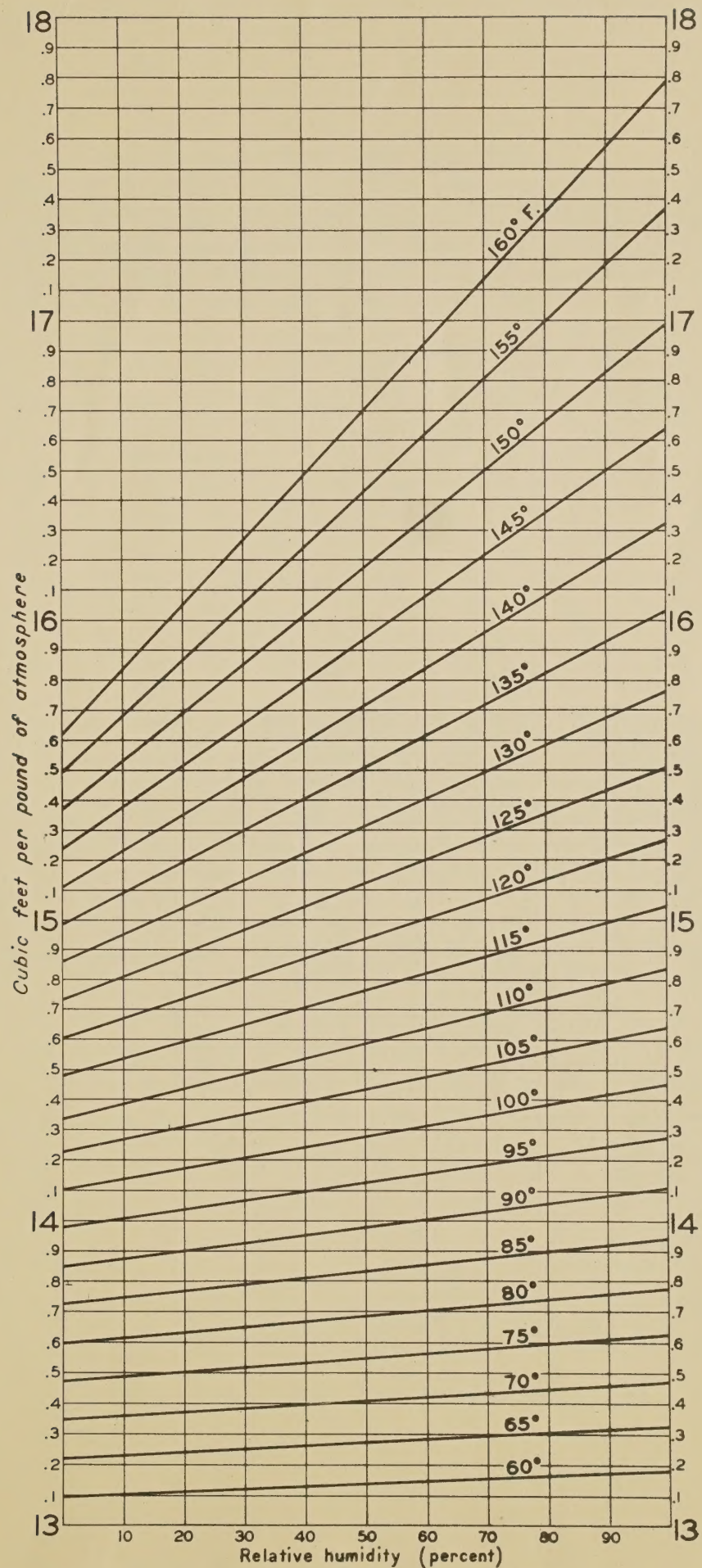


FIG.1-DENSITY OF ATMOSPHERE

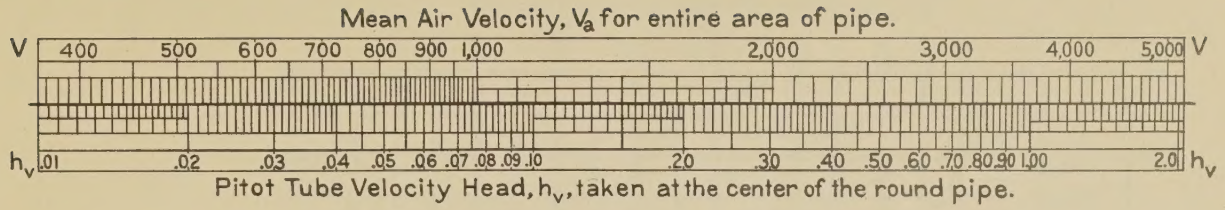


Fig. 2 - Scale of mean air velocities in round pipe in feet per minute

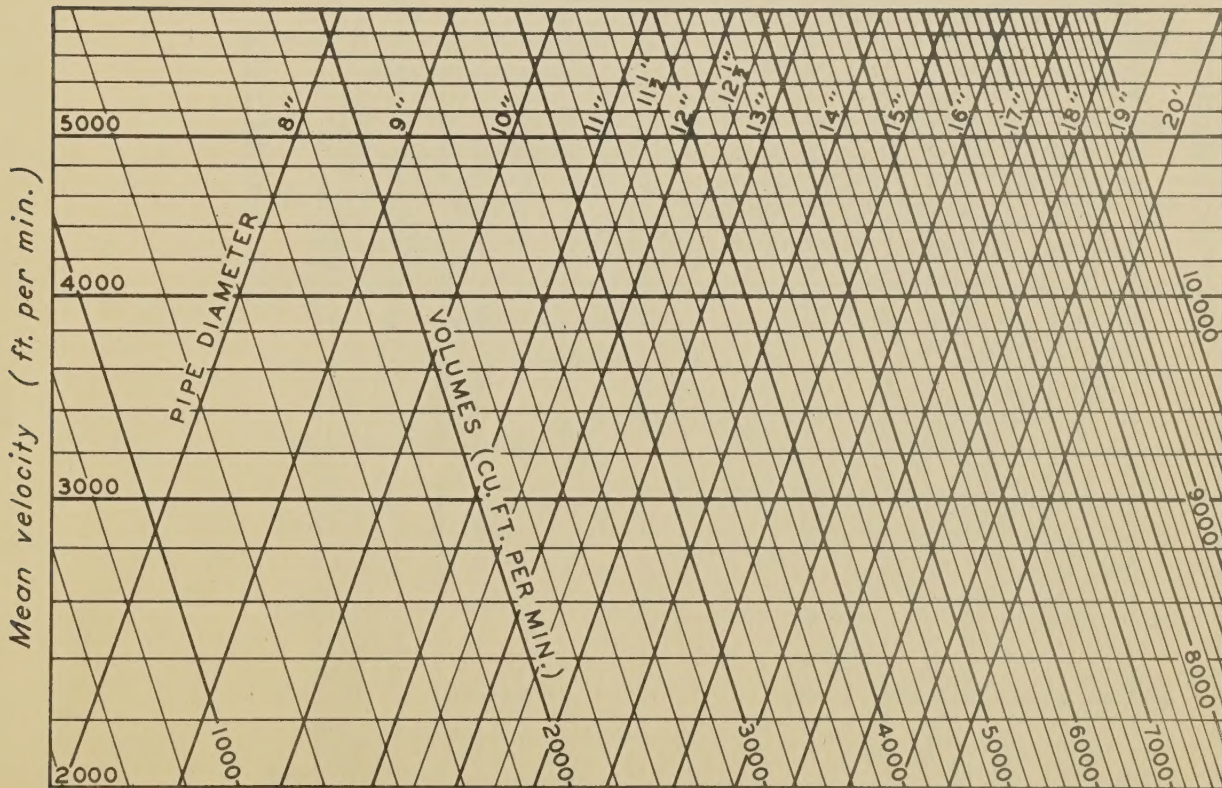


Fig. 3 - Approximate flow of air in cotton-gin piping

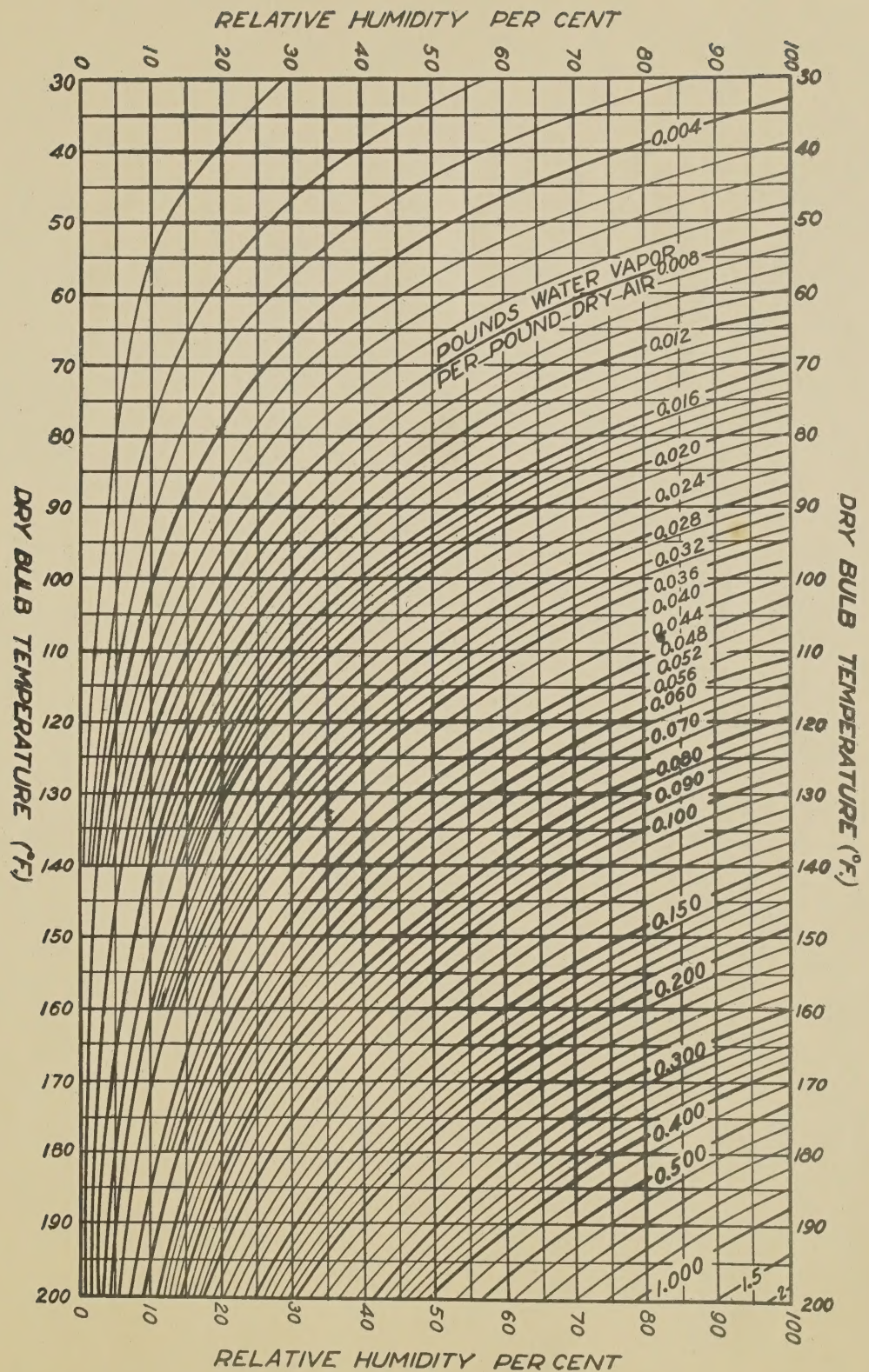


FIG. 4.—Pounds of water vapor per pound of dry air

